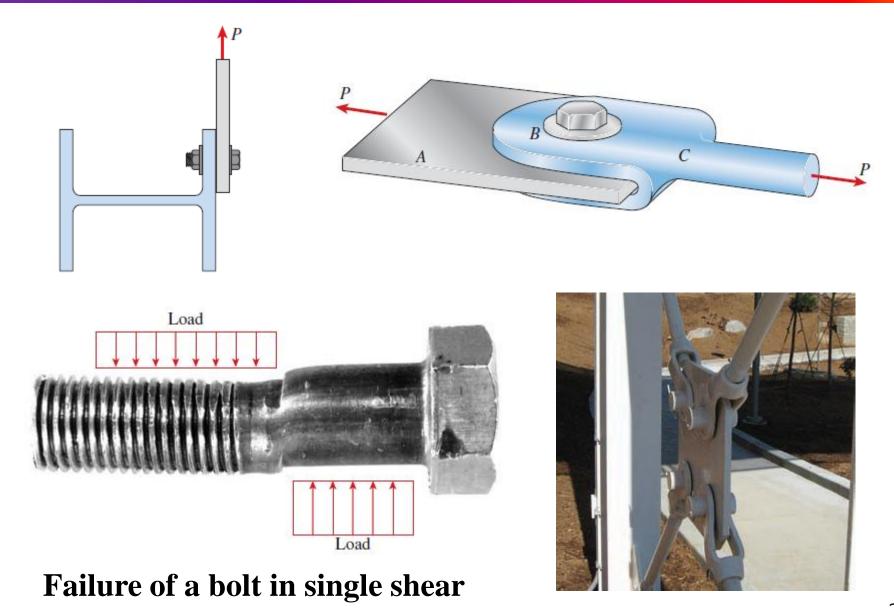
Shearing and Bearing Stress

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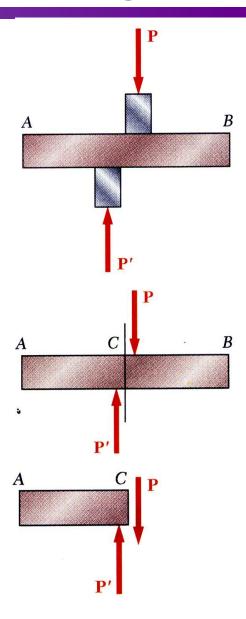
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Introduction to Pin Shearing and Bearing



Shearing Stress



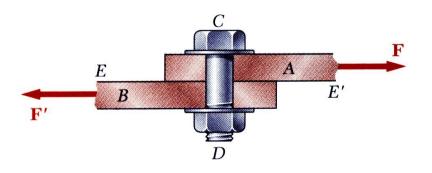
- Forces *P* and *P* ' are applied transversely to the member *AB*.
- Corresponding internal forces act in the plane of section *C* and are called *shearing* forces.
- The resultant of the internal shear force distribution is defined as the *shear* of the section and is equal to the load *P*.
- The corresponding average shearing stress is,

$$\tau_{\text{ave}} = \frac{P}{A}$$

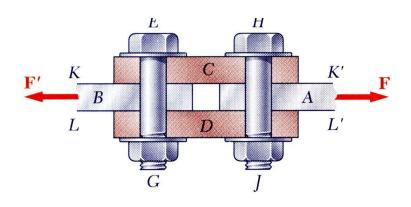
- Shearing stress distribution varies from zero at the member surfaces to maximum values that may be much larger than the average value.
- The shearing stress distribution cannot be assumed to be uniform.

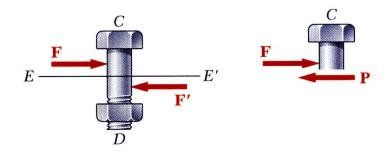
Shearing Stress

Single Shear

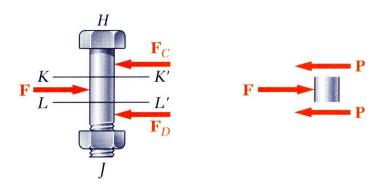


Double Shear



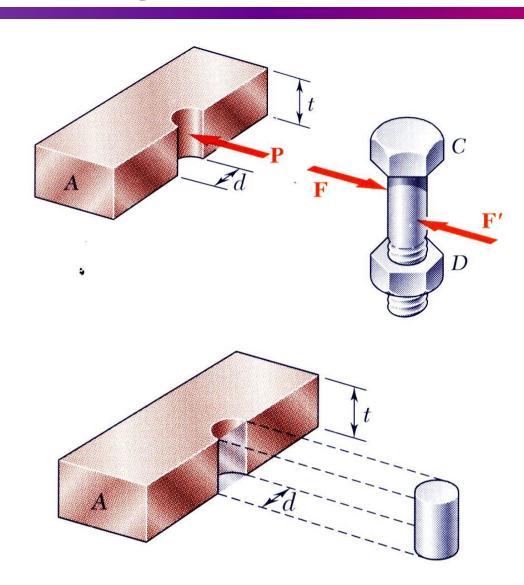


$$\tau_{\text{ave}} = \frac{P}{A} = \frac{F}{A}$$



$$\tau_{\text{ave}} = \frac{P}{A} = \frac{F}{2A}$$

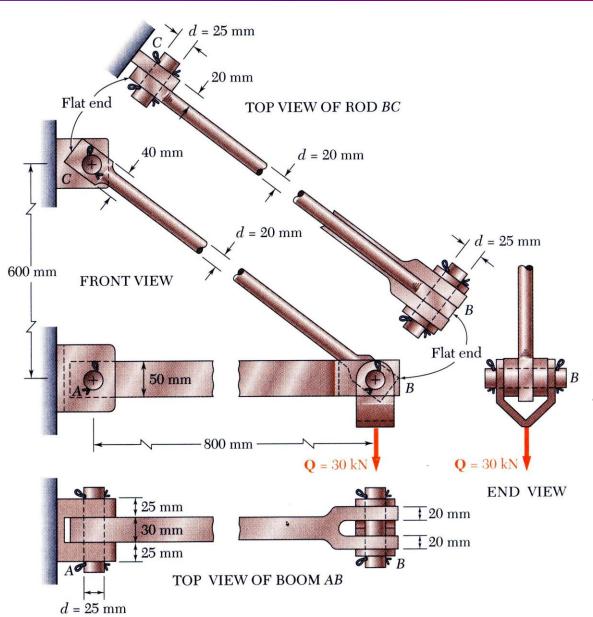
Bearing Stress



- Bolts, rivets, and pins create stresses on the points of contact or bearing surfaces of the members they connect.
- The resultant of the force distribution on the surface is equal and opposite to the force exerted on the pin.
- Corresponding average force intensity is called the bearing stress,

$$\sigma_{\rm b} = \frac{P}{A} = \frac{P}{t d}$$

Stress Analysis & Design Example

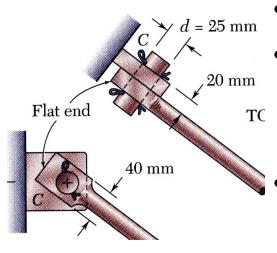


- Would like to determine the stresses in the members and connections of the structure shown.
- From a statics analysis: $F_{AB} = 40 \text{ kN}$ (compression)

$$F_{BC} = 50 \text{ kN (tension)}$$

• Must consider maximum normal stresses in *AB* and *BC*, and the shearing stress and bearing stress at each pinned connection

Rod & Boom Normal Stresses



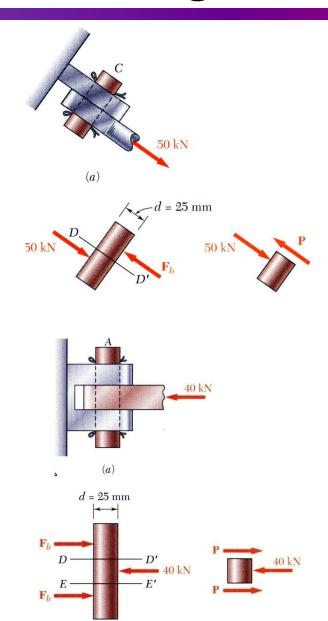
- The rod is in tension with an axial force of 50 kN.
- At the rod center, the average normal stress in the circular cross-section ($A = 314 \times 10^{-6} \text{m}^2$) is $\sigma_{BC} = +159 \text{ MPa}$.
- At the flattened rod ends, the smallest crosssectional area occurs at the pin centerline,

$$A = (20 \text{ mm})(40 \text{ mm} - 25 \text{ mm}) = 300 \times 10^{-6} \text{ m}^2$$

$$\sigma_{BC,end} = \frac{P}{A} = \frac{50 \times 10^3 N}{300 \times 10^{-6} \text{m}^2} = 167 \text{MPa}$$

- The boom is in compression with an axial force of 40 kN and average normal stress of –26.7 MPa.
- The minimum area sections at the boom ends are unstressed since the boom is in compression.

Pin Shearing Stresses



• The cross-sectional area for pins at *A*, *B*, and *C*,

$$A = \pi r^2 = \pi \left(\frac{25 \,\mathrm{mm}}{2}\right)^2 = 491 \times 10^{-6} \,\mathrm{m}^2$$

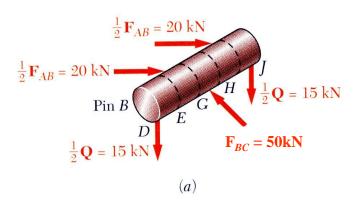
• The force on the pin at C is equal to the force exerted by the rod BC,

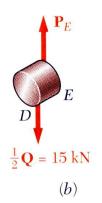
$$\tau_{C,ave} = \frac{P}{A} = \frac{50 \times 10^3 \,\text{N}}{491 \times 10^{-6} \,\text{m}^2} = 102 \,\text{MPa}$$

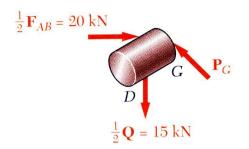
• The pin at *A* is in double shear with a total force equal to the force exerted by the boom *AB*,

$$\tau_{A,ave} = \frac{P}{A} = \frac{20 \text{kN}}{491 \times 10^{-6} \text{m}^2} = 40.7 \text{ MPa}$$

Pin Shearing Stresses







• Divide the pin at *B* into sections to determine the section with the largest shear force,

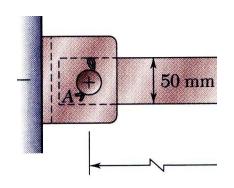
$$P_E = 15 \text{kN}$$

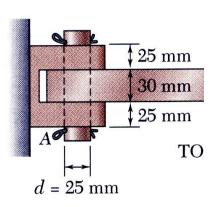
 $P_G = 25 \text{kN (largest)}$

• Evaluate the corresponding average shearing stress,

$$\tau_{B,ave} = \frac{P_G}{A} = \frac{25 \text{kN}}{491 \times 10^{-6} \text{m}^2} = 50.9 \text{ MPa}$$

Pin Bearing Stresses



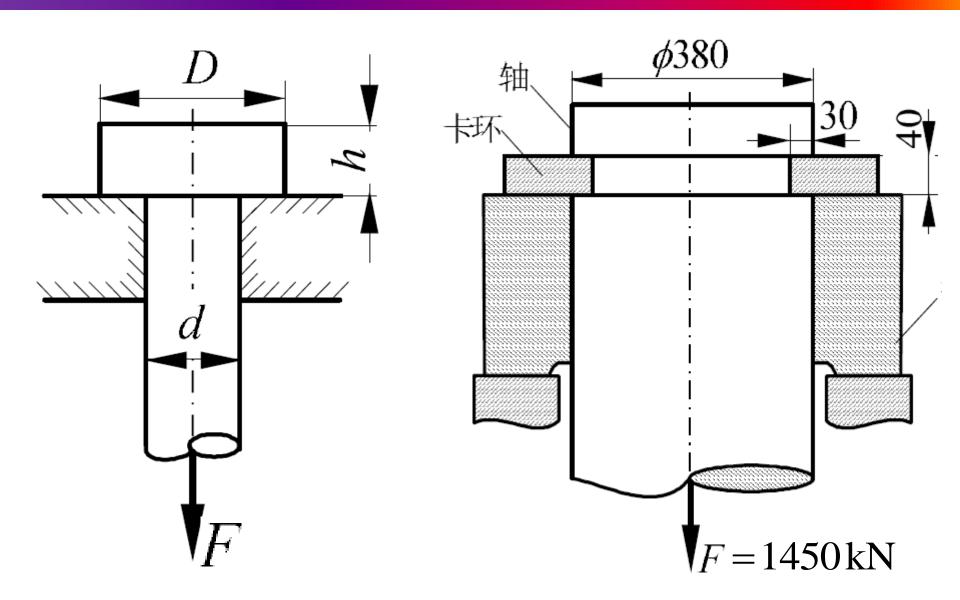


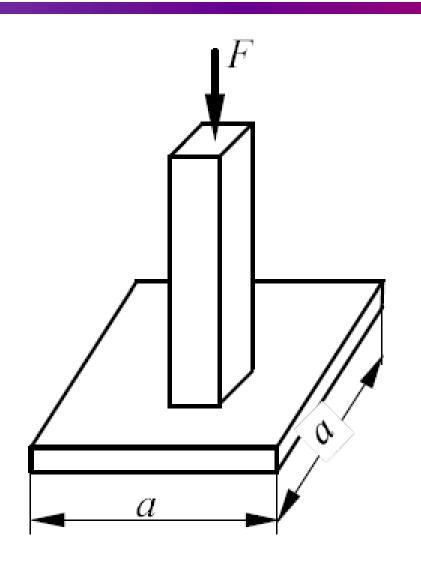
• To determine the bearing stress at A in the boom AB, we have t = 30 mm and d = 25 mm,

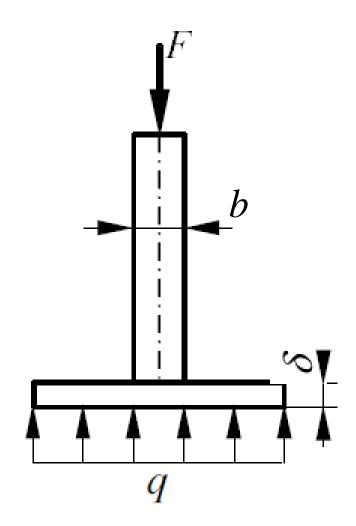
$$\sigma_b = \frac{P}{td} = \frac{40 \text{kN}}{(30 \text{mm})(25 \text{mm})} = 53.3 \text{MPa}$$

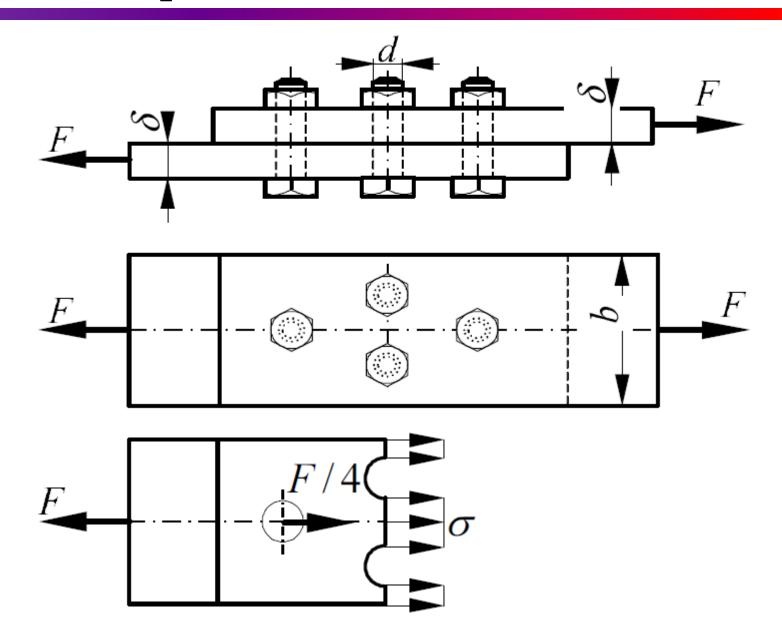
• To determine the bearing stress at A in the bracket, we have t = 2(25 mm) = 50 mm and d = 25 mm,

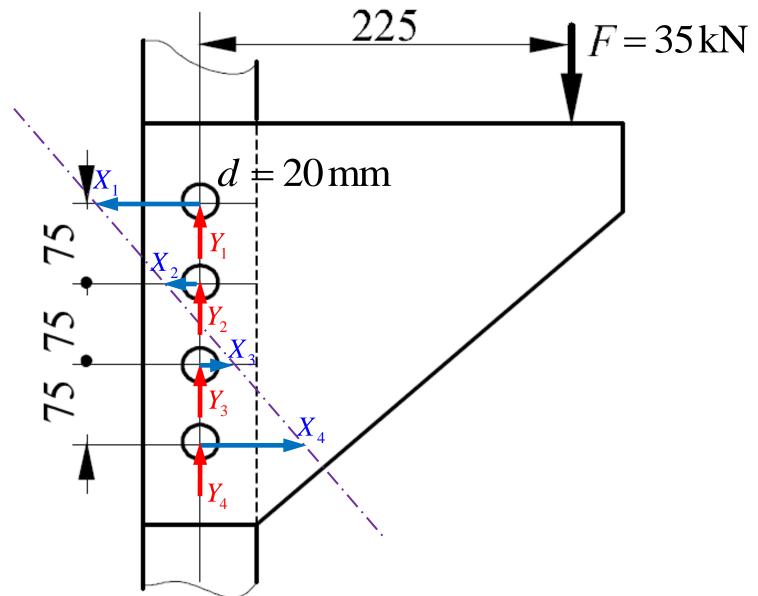
$$\sigma_b = \frac{P}{td} = \frac{40 \text{kN}}{(50 \text{mm})(25 \text{mm})} = 32.0 \text{MPa}$$











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